

Control Systems I

Loop Shaping

Colin Jones

Laboratoire d'Automatique

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Sensitivity & Complementary Sensitivity

Loop Shaping

Goal

Design a controller to achieve a set of specifications on the closed-loop system

Challenge

Closed-loop transfer functions are a highly nonlinear function of the control law

$$\mathcal{T} = \frac{GK}{1 + GK} \qquad \mathcal{S} = \frac{1}{1 + GK}$$

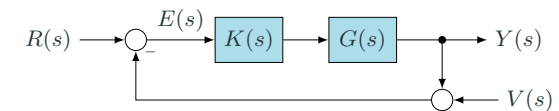
Idea

Define closed-loop characteristics in terms of open-loop response GK .

Shaping the response GK is *linear* in K , and much easier

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Recall: Closed-Loop Transfer Functions



Two quantities that define the performance of the system:

- Response of error $E(s)$ to output noise $V(s)$

$$\mathcal{S}(s) := \frac{E(s)}{V(s)} = \frac{1}{1 + G(s)K(s)} \quad \text{Sensitivity function}$$

- Response of output $Y(s)$ to reference $R(s)$

$$\mathcal{T}(s) := \frac{Y(s)}{R(s)} = \frac{G(s)K(s)}{1 + G(s)K(s)} \quad \text{Complementary sensitivity function}$$

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Sensitivity & Complementary Sensitivity Functions

Sensitivity Function

$$\mathcal{S}(s) = \frac{E(s)}{V(s)} = \frac{1}{1 + G(s)K(s)}$$

- Impact of noise on the error
- Ideal value : 0

Complementary Sensitivity Function

$$\mathcal{T}(s) = \frac{Y(s)}{Y_c(s)} = \frac{G(s)K(s)}{1 + G(s)K(s)}$$

- Impact of reference on the output
- Ideal value : 1

Functions are *complementary*:

$$\mathcal{S}(s) + \mathcal{T}(s) = \frac{1}{1 + G(s)K(s)} + \frac{G(s)K(s)}{1 + G(s)K(s)} = 1$$

Changes in one will cause changes in the other - limits of performance

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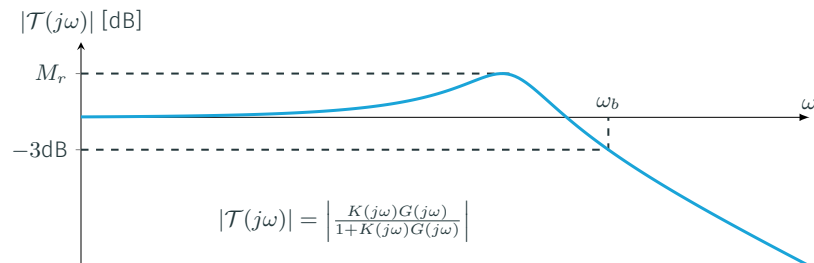
Frequency Response

The sensitivity and complementary sensitivity functions are transfer functions:

- We can compute their frequency responses: $\mathcal{S}(j\omega)$, $\mathcal{T}(j\omega)$
- These describe the response of the system in terms of disturbance rejection and tracking performance
- By shaping these, we can design a system with desired behaviour
- Complementarity represents an inherent tradeoff: tracking vs noise rejection
- Idea: Good tracking and noise rejection at low frequencies, bad at high

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Complementary Sensitivity Function

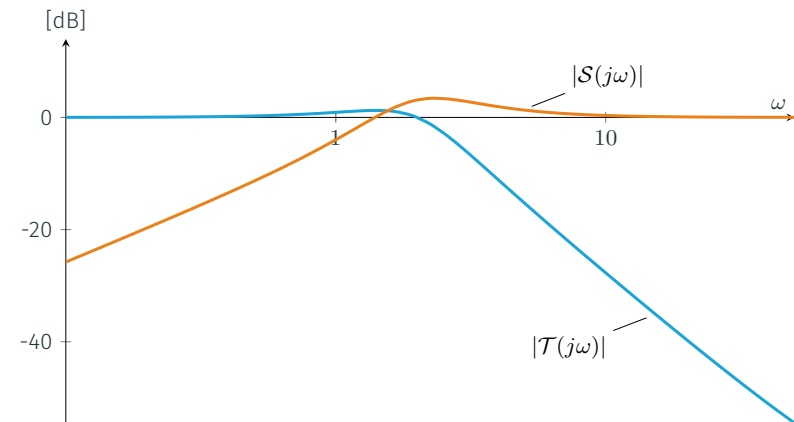


Desired shape:

- Low-frequency gain of 0 dB
- Small resonance peak M_r at the resonant frequency ω_r
- Large bandwidth defined by the pass-band $[0, \omega_b]$, and the cutoff-frequency ω_b
- High roll-off after ω_b to make the system insensitive to measurement noise, and unmodeled dynamics

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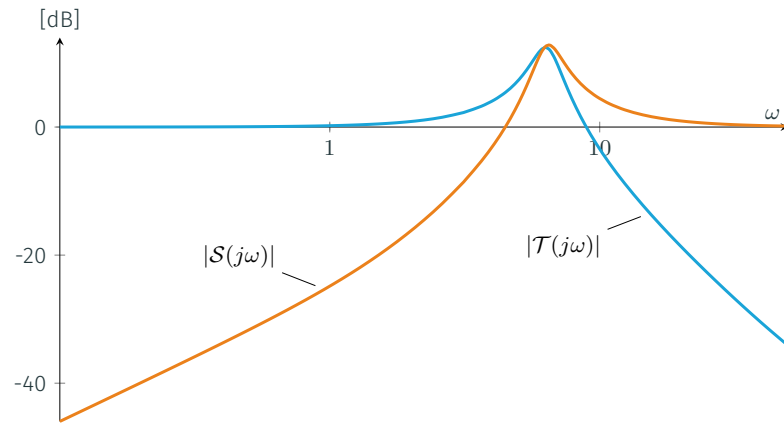
Example



Low gain / high stability margin

7

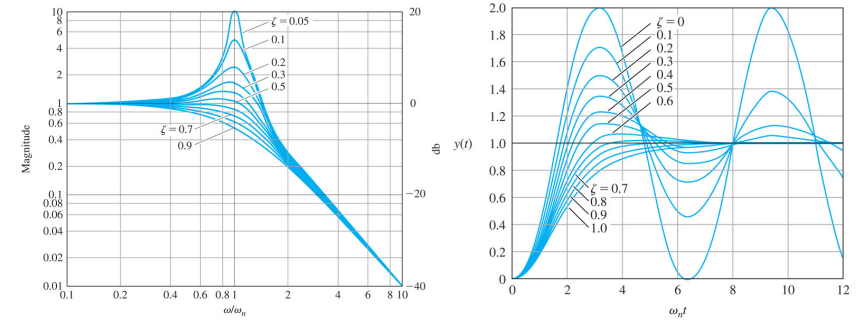
Example



High gain / low stability margin

8

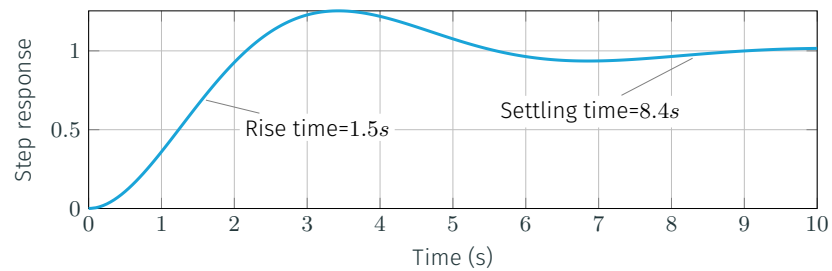
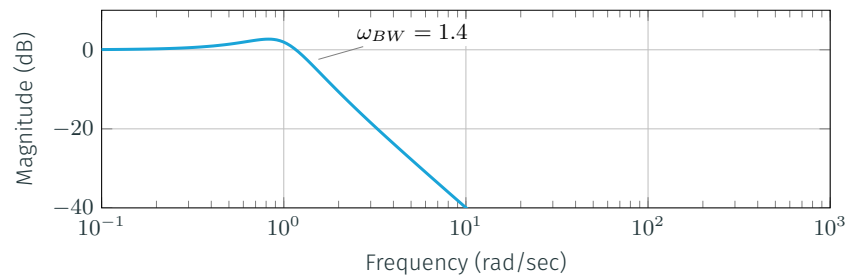
Relation to Time-Domain Behaviours



Magnitude of resonant peak related to the damping of the closed-loop system.

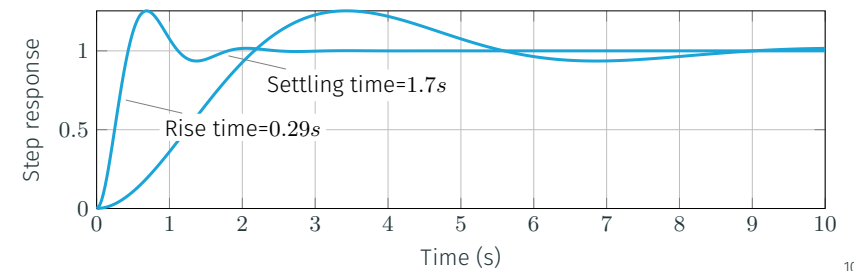
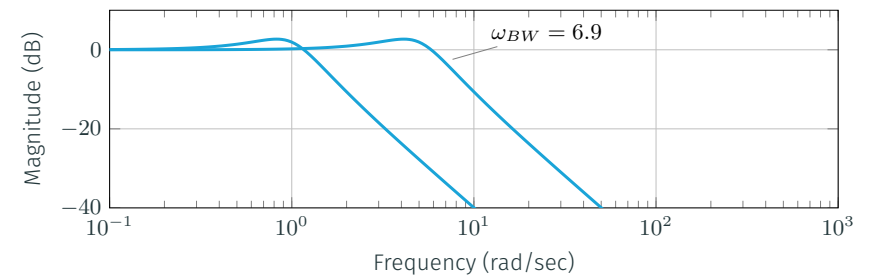
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Bandwidth Defines Rise Time & Settling Time



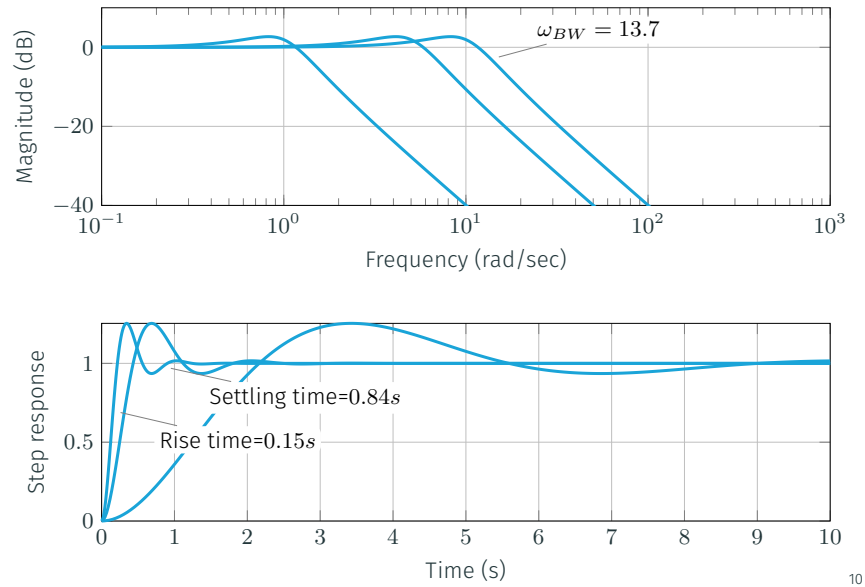
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Bandwidth Defines Rise Time & Settling Time



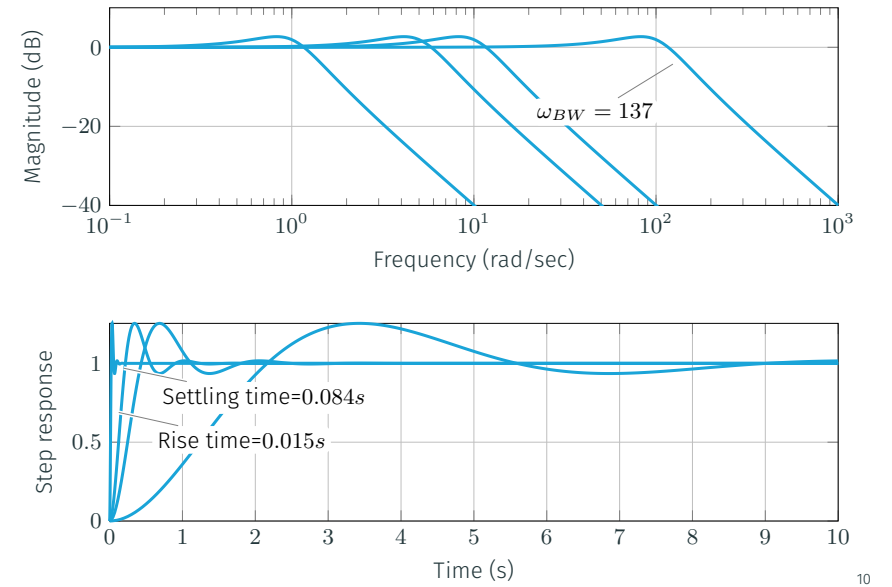
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Bandwidth Defines Rise Time & Settling Time



Example 9.2

Bandwidth Defines Rise Time & Settling Time



Loop Shaping

Open-Loop Properties $\Leftrightarrow$ Closed-Loop Properties

$$\mathcal{T}(j\omega) = \frac{K(j\omega)G(j\omega)}{1 + K(j\omega)G(j\omega)} = 1 - \frac{1}{1 + K(j\omega)G(j\omega)}$$

$$\mathcal{T}(j\omega) = 1 \text{ for small } \omega \quad \Leftrightarrow \quad K(j\omega)G(j\omega) \text{ large for small } \omega$$

→ Integrator (pole at 0)

$$|\mathcal{T}(j\omega)| \ll 0dB \text{ for large } \omega \quad \Leftrightarrow \quad |K(j\omega)G(j\omega)| \ll 0dB \text{ for large } \omega$$

$$\text{Low resonance peak} \quad \Leftrightarrow \quad \text{Large stability margins}$$

→ Resonance when $|1 + K(j\omega_r)G(j\omega_r)|$ is small

→ $K(j\omega_r)G(j\omega_r) \approx -1$

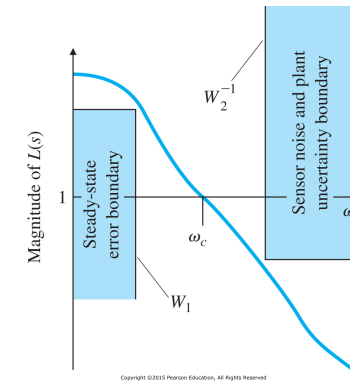
$$\text{Specified rise time/settling time} \quad \Leftrightarrow \quad \text{Crossover frequency}$$

→ Open-loop crossover frequency $\approx$ Closed-loop bandwidth

Can describe good closed-loop behaviour via open-loop frequency response

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Loopshaping Goals



(Note: $L(s) = K(s)G(s)$ is the **Loop gain**)

Low-frequency slope (system type) and gain are chosen for steady-state error

High-frequency roll-off is determined by actuator/ sensor limitations and system bandwidth goals.

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Closed-loop Bandwidth $\approx$ Crossover Frequency

The open-loop frequency response has been designed for

$$|KG(j\omega)| \gg 1 \text{ for } \omega \ll \omega_c$$

$$|KG(j\omega)| \ll 1 \text{ for } \omega \gg \omega_c$$

The closed-loop response is therefore

$$|\mathcal{T}(j\omega)| = \left| \frac{KG(j\omega)}{1 + KG(j\omega)} \right| \approx \begin{cases} 1, & \omega \ll \omega_c \\ |KG|, & \omega \gg \omega_c \end{cases}$$

Around crossover, we have $|KG(j\omega)| \approx 1$ and $\mathcal{T}(j\omega)$ depends on the phase margin

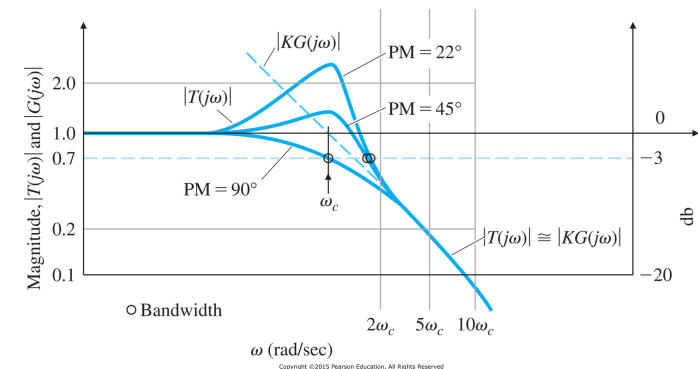
$$KG(j\omega_c) = e^{j(\pi - \phi)} = -e^{-j\phi}$$

$$|\mathcal{T}(j\omega_c)| = \left| \frac{KG(j\omega_c)}{1 + KG(j\omega_c)} \right| = \left| \frac{-e^{-j\phi}}{1 - e^{-j\phi}} \right|$$

If $\phi = 90^\circ$, then $|\mathcal{T}(j\omega_c)| = 0.707 = -3dB$

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Closed-loop Bandwidth $\approx$ Crossover Frequency



Closed loop bandwidth is within a factor of two of the crossover frequency

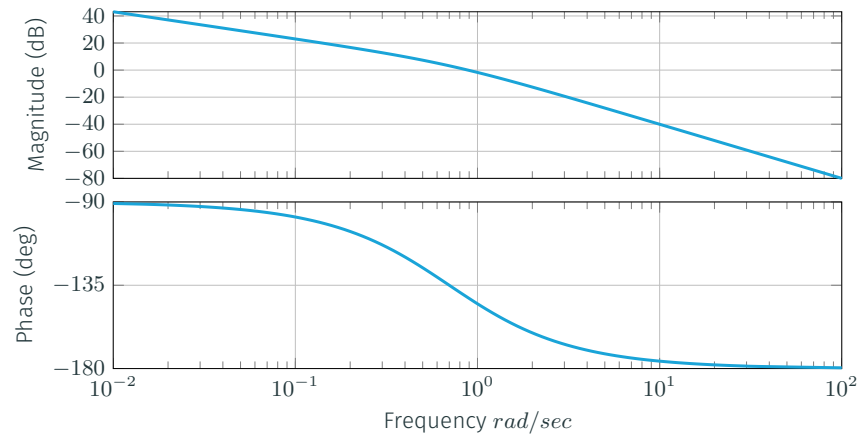
$$\omega_c \leq \omega_{BW} \leq 2\omega_c$$

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Resonance and Phase Margin

Consider the prototype open-loop model

$$G(s) = \frac{\omega_n^2}{s(s + 2\zeta\omega_n)}$$



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Resonance and Phase Margin

Consider the prototype open-loop model

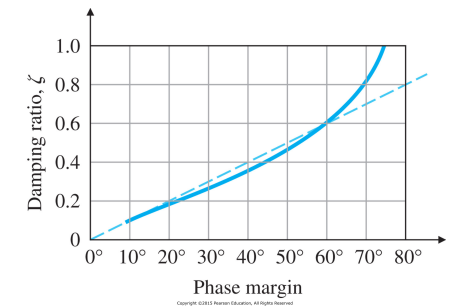
$$G(s) = \frac{\omega_n^2}{s(s + 2\zeta\omega_n)}$$

With unity feedback, we get the closed-loop system

$$\mathcal{T}(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

$$PM = \tan^{-1} \left[\frac{2\zeta}{\sqrt{\sqrt{1 + 4\zeta^4} - 2\zeta^2}} \right]$$

$$\approx 100\zeta$$

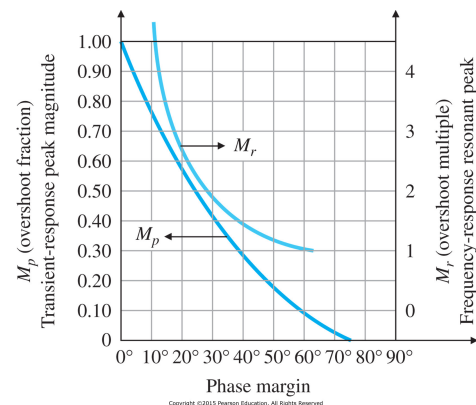


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Resonance and Phase Margin

Consider the prototype open-loop model

$$G(s) = \frac{\omega_n^2}{s(s + 2\zeta\omega_n)}$$



Damping ratio determines the step response overshoot, and the size of the resonant peak.

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Loopshaping Goals

- KG large for small ω (Steady-state error)
- KG small for large ω (Modeling errors, etc)
- Crossover frequency chosen according to desired closed-loop bandwidth
- Good stability margins

Goal: Choose $K(s)$ to satisfy these requirements

Tools:

- Overall gain: Moves magnitude plot up and down
- Lead compensator
- Lag compensator

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Lead Compensator

Phase Lead Compensator

Lead Compensator

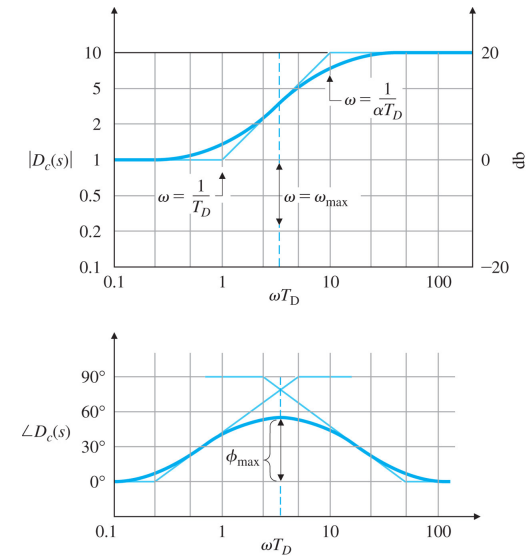
$$D_c(s) := \frac{T_D s + 1}{\alpha T_D s + 1} \quad \alpha < 1$$

Within interval of interest

- Phase increased by $\phi_{\max}$
- Slope increased by 20dB/dec

Utility:

- Place near crossover frequency to increase phase



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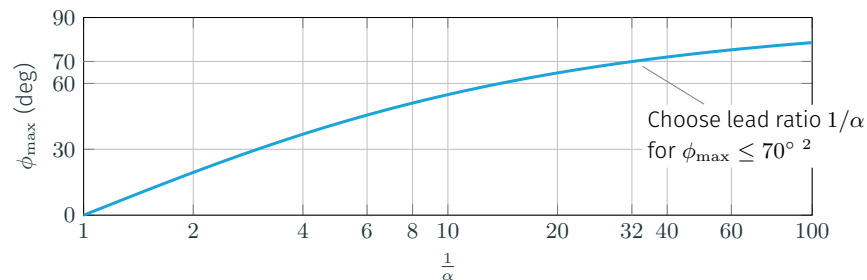
How much is the phase increased?

Max phase increase happens at the center of the pole and zero¹

$$\omega_{\max} = \frac{1}{T_D \sqrt{\alpha}} \quad \log \omega_{\max} = \frac{1}{2} \left(\log \frac{1}{T_D} + \log \frac{1}{\alpha T_D} \right)$$

The amount of phase lead at this point is

$$\sin \phi_{\max} = \frac{1 - \alpha}{1 + \alpha} \quad \alpha = \frac{1 - \sin \phi_{\max}}{1 + \sin \phi_{\max}}$$



¹See Problem 6.44

²Or gain at high frequencies may be too much, and multiple lead compensators should be used.

Example

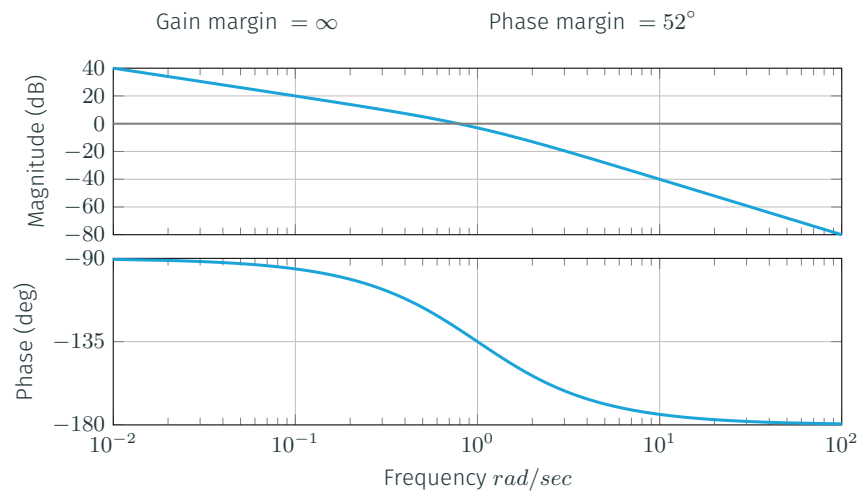
Consider the following system

$$G(s) = \frac{1}{s(s+1)}$$

Requirements:

- Steady-state error less than 0.1 in response to a ramp reference
- Overshoot of less than $M_P \leq 25\%$

Example



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Example

Steady-state error less than 0.1 in response to a ramp reference

This is a Type 1 system:

→ Error with respect to a ramp input is $\frac{1}{\gamma}$

$$\gamma = \lim_{s \rightarrow 0} sG(s) = \lim_{s \rightarrow 0} \frac{1}{s+1} = 1$$

Steady-state error in response to a ramp is $e_{ss} = 1$.

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Example

Steady-state error less than 0.1 in response to a ramp reference

This is a Type 1 system:

→ Error with respect to a ramp input is $\frac{1}{\gamma}$

$$\gamma = \lim_{s \rightarrow 0} sG(s) = \lim_{s \rightarrow 0} \frac{1}{s+1} = 1$$

Steady-state error in response to a ramp is $e_{ss} = 1$.

Try the simplest controller to improve this: Proportional gain K

Loop gain is now $L(s) = KG(s) = \frac{K}{s(s+1)}$.

$$\gamma = \lim_{s \rightarrow 0} sKG(s) = \lim_{s \rightarrow 0} \frac{K}{s+1} = K$$

Steady-state error in response to a ramp is $e_{ss} = \frac{1}{K}$.

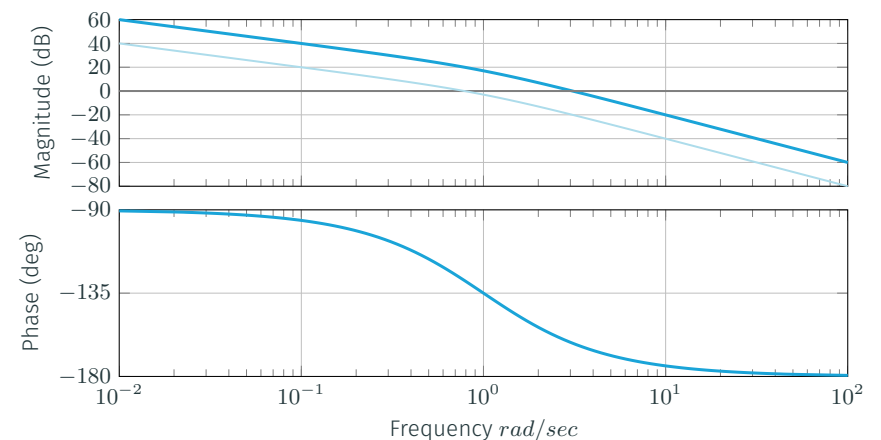
→ Choose $K = 10$

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Example

Gain margin = ∞

Phase margin = 18°



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Example

Overshoot of less than $M_P \leq 25\%$

From Slide 16 we see that a phase margin of 45° will do

→ Add a phase lead compensator

Current phase margin is $\approx 20^\circ$ → Requires an increase of 25°

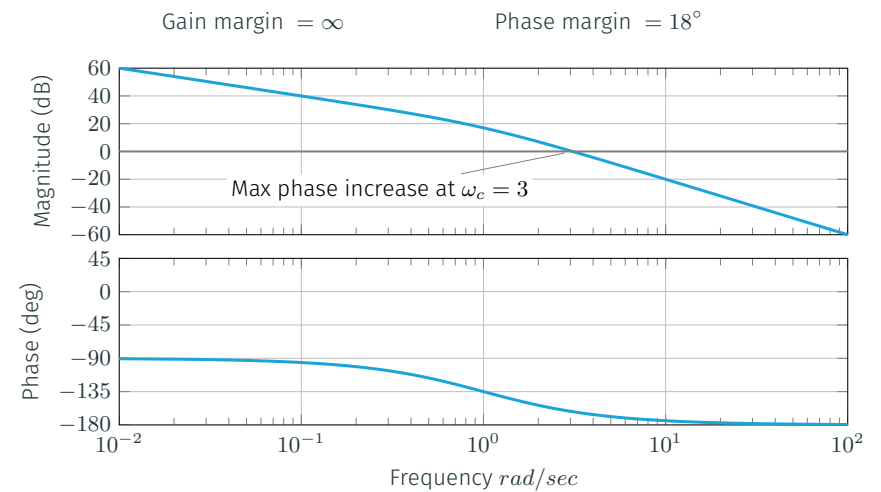
Lead compensator also increases gain → Increases crossover frequency

Increase phase by $\approx 40^\circ$ to compensate

Slide 19 shows $\alpha = 1/5$ will increase phase by $\approx 40^\circ$

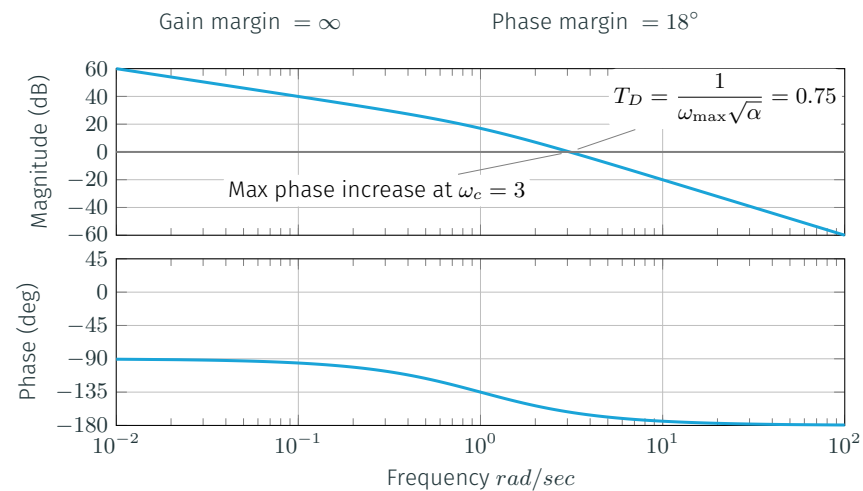
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Example



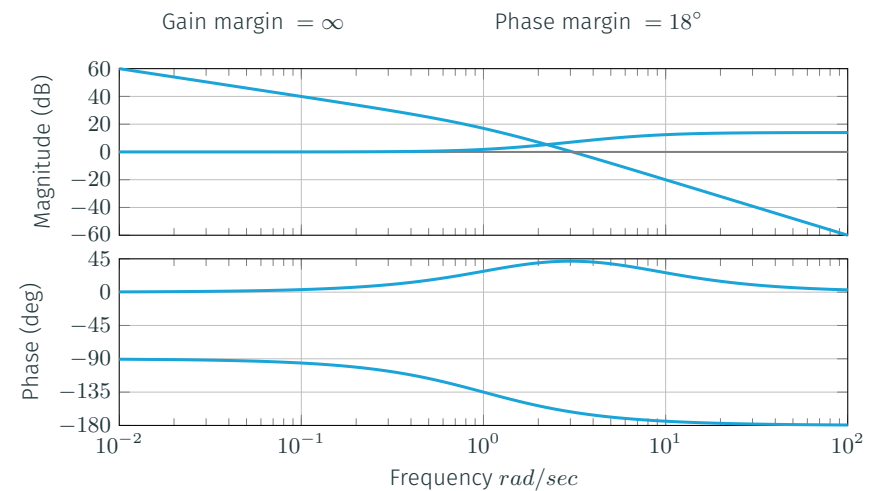
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Example



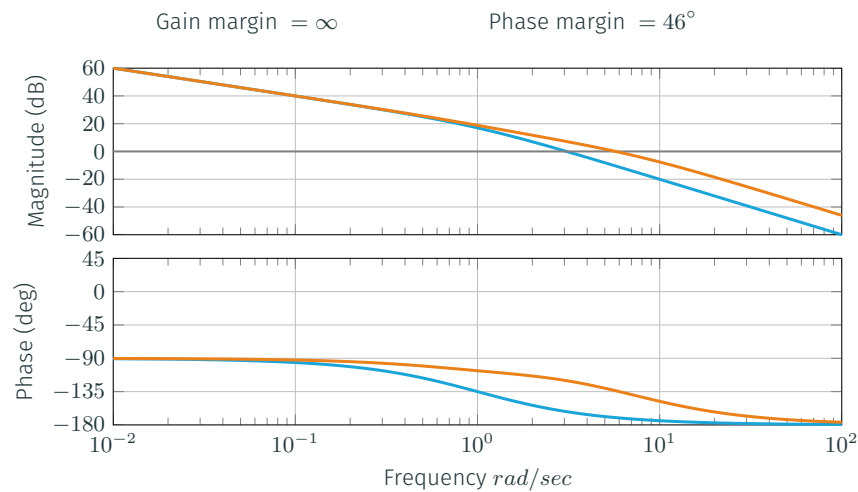
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Example



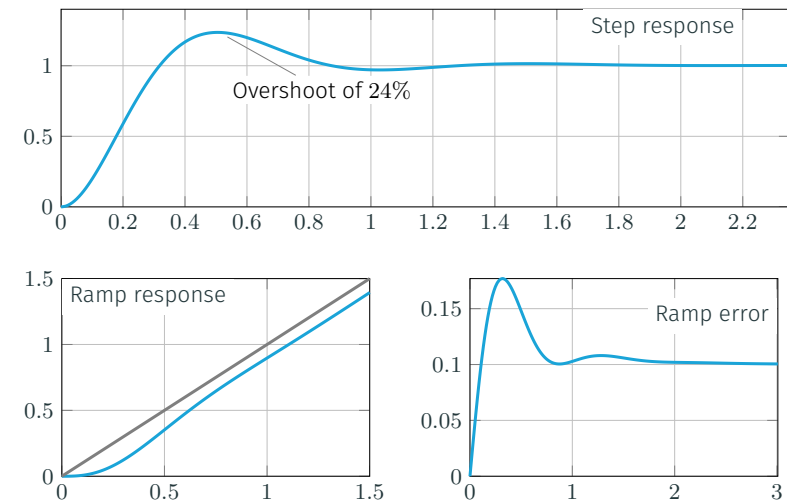
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Example



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Example - Time Domain Result $G(s) = \frac{1}{s(s+1)}$ $K(s) = 10 \frac{0.75s+1}{0.15s+1}$



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Lead Design Summary

Generally three criteria

1. Crossover frequency $\leftarrow$ Bandwidth, rise time and settling time
2. Phase margin $\leftarrow$ Damping coefficient ζ and overshoot M_p
3. Low-frequency gain $\leftarrow$ Steady-state error

Design procedure

1. Choose system type and controller gain K such that
 - steady-state gain targets are met
 - open-loop crossover frequency is a factor of two below the desired closed-loop bandwidth
2. Determine the increase in phase margin required (add about 10° to compensate for bandwidth increase) and choose α to give the desired increase.
3. Choose $\omega_{\max}$ to be the crossover frequency, and set $T_D = \frac{1}{\omega_{\max} \sqrt{\alpha}}$

Note that this procedure may require customization for any particular system.

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Quick and Dirty Using Bode's Gain-Phase Relationship

Main idea: A low slope at crossover provides a good phase margin.
e.g., -20dB/dec gives a phase margin of about 90°

Slope must be constant for a decade around the crossover frequency for approximation to hold. Equivalent to choosing $1/\alpha = \sqrt{5} \approx 3$.

$$D_c(s) := \frac{3s + \omega_c}{s/3 + \omega_c}$$

Ignore the phase plot, and work only with the magnitude plot.

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Gain-Phase Relationship

Bode Gain-Phase Theorem

For any stable minimum-phase system (i.e., one with no RHP zeros or poles), the phase of $G(j\omega)$ is uniquely related to the magnitude of $G(j\omega)$

$$\angle G(j\omega_0) = \frac{1}{\pi} \int_{-\infty}^{\infty} \left(\frac{dM}{du} \right) W(u) du \quad (\text{in radians})$$

where

- $M = \log \text{Magnitude} = \ln |G(j\omega)|$
- $u = \text{normalized frequency} = \ln(\omega/\omega_0)$
- $W(u) = \text{weighting function} = \ln(\coth|u|/2)$

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Gain-Phase Relationship

Bode Gain-Phase Theorem (Simple form)

$$\angle G(j\omega) \approx n \times 90^\circ$$

where n is the slope of $|G(j\omega)|$ in units of decade of amplitude per decade of frequency.

If the crossover frequency is ω_0 , i.e., the gain is $|K(j\omega_0)G(j\omega_0)| = 1$, then

- $\angle G(j\omega_0) \approx -90^\circ$ if $n = -1$ (-20dB / dec)
- $\angle G(j\omega_0) \approx -180^\circ$ if $n = -2$ (-40dB / dec)

Main idea: A low slope at crossover provides a good phase margin.

e.g., -20dB/dec gives a phase margin of about 90°

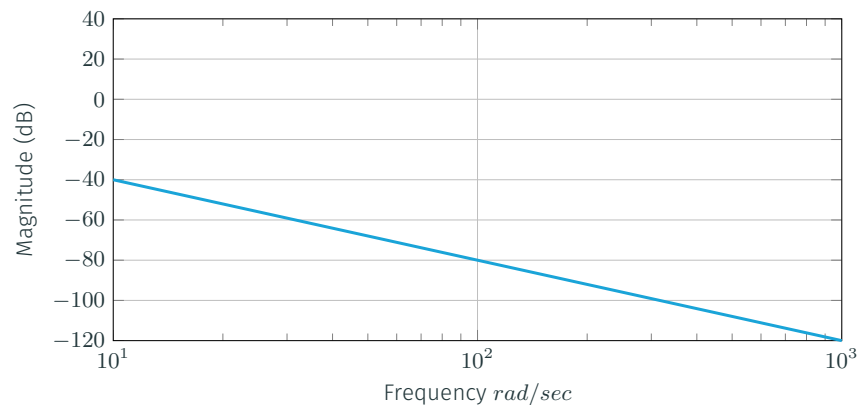
²Slope must be constant for a decade around the crossover frequency for approximation

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Simple Example

$$G(s) = \frac{1}{s^2}$$

Design a lead compensator for the system providing zero steady-state error in response to a ramp input, around 60° phase margin and a bandwidth of at least 100r/s .

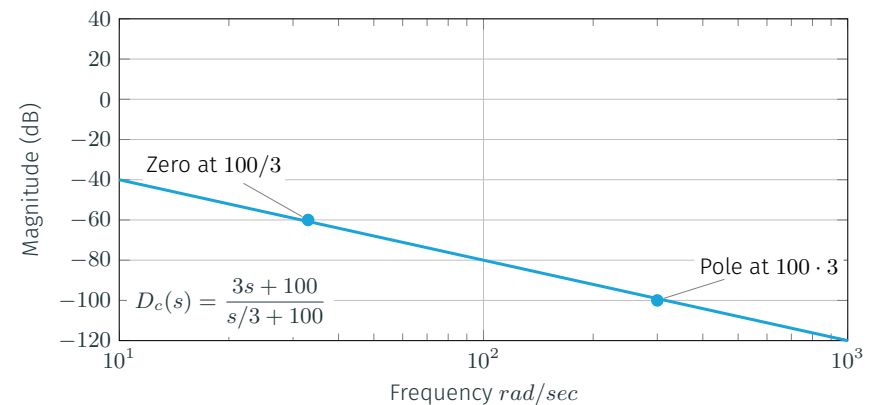


31

Simple Example

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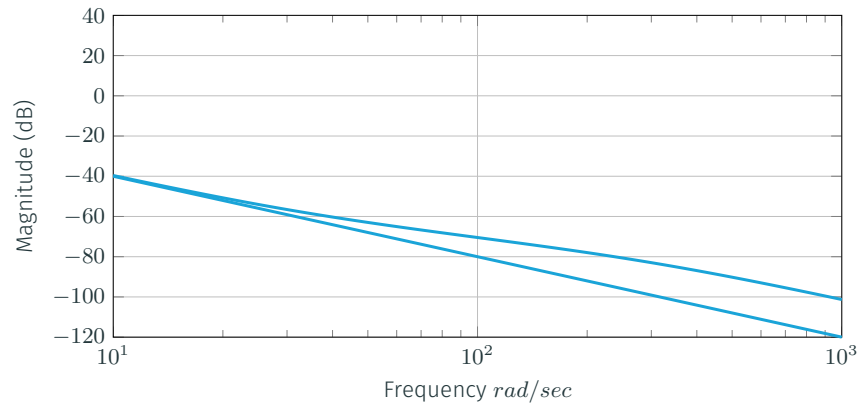


31

Simple Example

$$G(s) = \frac{1}{s^2}$$

Design a lead compensator for the system providing zero steady-state error in response to a ramp input, around 60° phase margin and a bandwidth of at least $100r/s$.

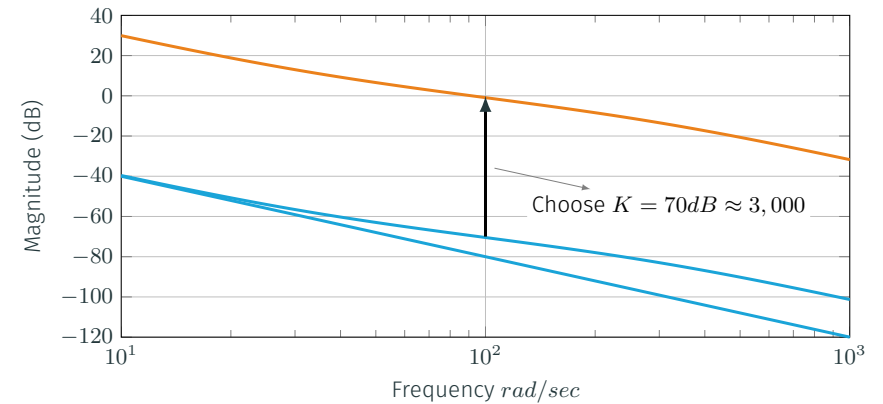


31

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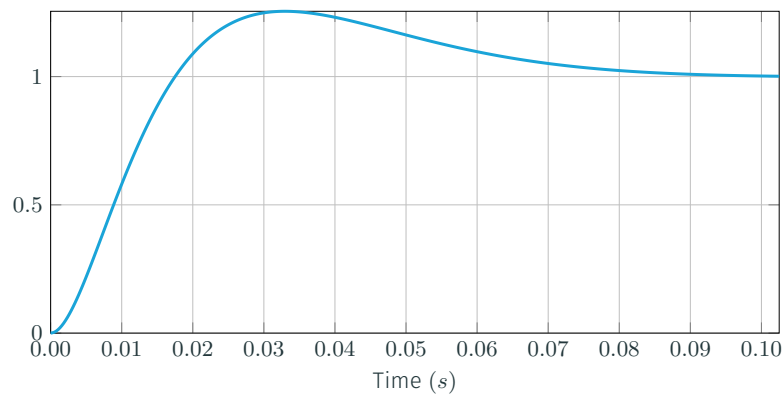


31

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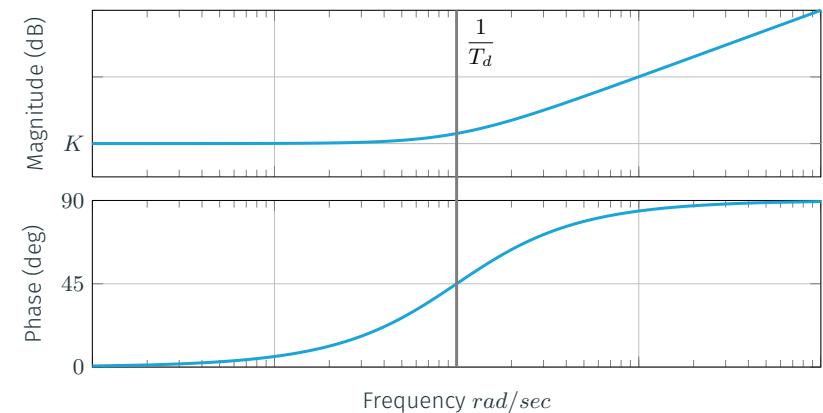
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Interpretation of PD Controller

Consider the PD controller:

$$K(s) = K(1 + T_D s)$$

This is a lead compensator with the pole at $s = -\infty$, or $\alpha = 0$.



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Example - PD Controller

$$G(s) = 0.05 \frac{80 - s}{s(s + 2.05)}$$

Design a PD controller such that:

- Steady-state error to step input is zero
- Track ramp with steady-state error less than 0.05
- Closed-loop step response with time-constant less than 0.07s
- Phase margin greater than 60°

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Steady-State Error

- Steady-state error to step input is zero
- Track ramp with steady-state error less than 0.05

Consider a proportional controller: $K(s) = K$

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Steady-State Error

- Steady-state error to step input is zero
- Track ramp with steady-state error less than 0.05

Consider a proportional controller: $K(s) = K$

$$K(s)G(s) = K \cdot 0.05 \frac{80 - s}{s(s + 2.05)}$$

Type 1 system

- Zero steady-state error to a step
- Steady-state error to a ramp reference $r(t) = t$ is $1/\gamma$

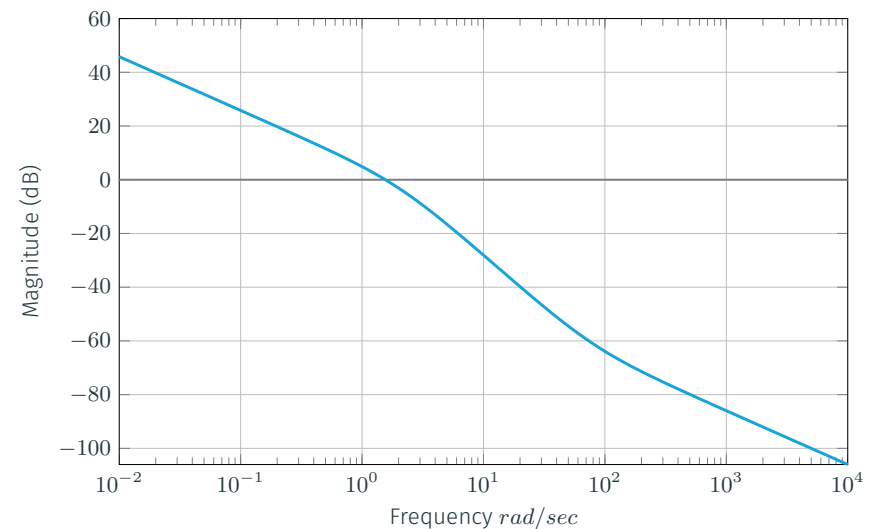
$$\gamma := \lim_{s \rightarrow 0} s^q K(s)G(s) = K \frac{B(0)}{A(0)} = K \cdot 0.05 \frac{80}{2.05} = K \cdot 1.9$$

$$\text{Error} = 1/(K \cdot 1.9) \leq 0.05$$

$$\text{Therefore, the gain } K \geq 1/(0.05 \cdot 1.9) = 10.5$$

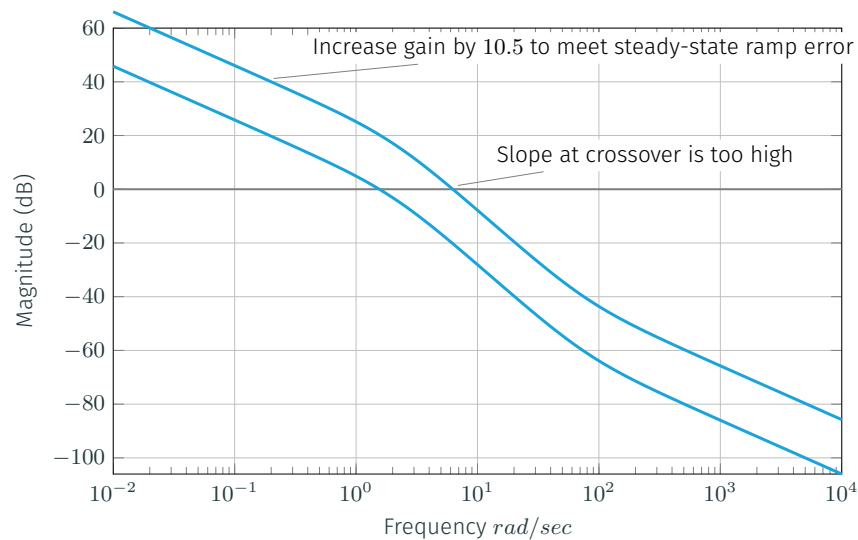
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Frequency Response



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Frequency Response



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Add Lead Compensator

Goal: Improve phase margin

Add derivative term (Lead compensator)

$$K(s) = 10.5(1 + T_D s)$$

How to choose T_D ?

- Sets bandwidth of the system
- Roughly sets the time constant of the closed-loop step response

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Bandwidth and Time Constant

Assume: Phase margin of about 90° at a crossover frequency of ω_c

For frequencies near ω_c , the open-loop gain is approximately:

$$K(j\omega)G(j\omega) \approx \frac{\omega_c}{j\omega}$$

The step response is approximately:

$$Y(s) = \frac{K(s)G(s)}{1 + K(s)G(s)} \cdot \frac{1}{s} \approx \frac{\frac{\omega_c}{s}}{1 + \frac{\omega_c}{s}} \cdot \frac{1}{s} = \frac{\omega_c}{s + \omega_c} \cdot \frac{1}{s} = \frac{-1}{s + \omega_c} + \frac{1}{s}$$

Gives the time response:

$$y(t) = 1 - e^{-\omega_c t}$$

The system time constant is approximately $1/\omega_c$

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Add Lead Compensator

Goal: Improve phase margin

Add derivative term (Lead compensator)

$$K(s) = 10.5(1 + T_D s)$$

How to choose T_D ?

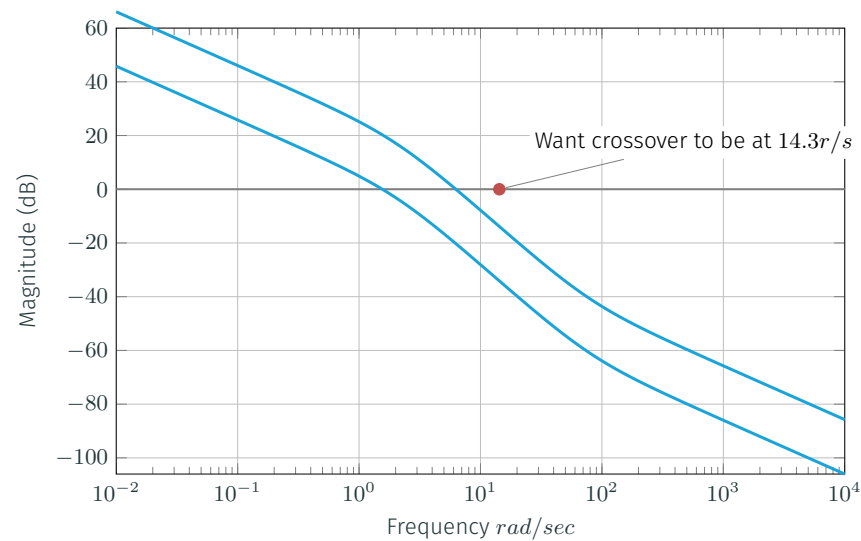
- Sets bandwidth of the system
- Roughly sets the time constant of the closed-loop step response

Goal: Time constant less than 0.07s

Choose $\omega_c \geq 1/0.07 = 14.3$

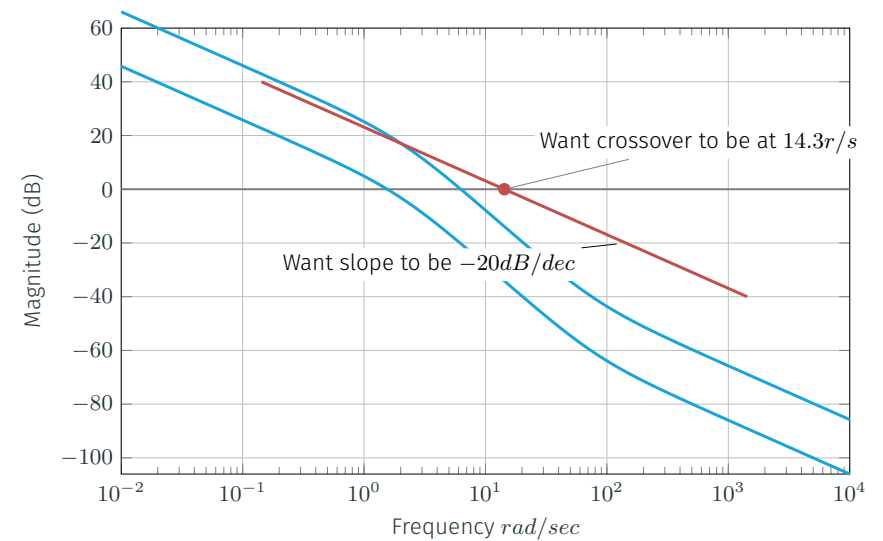
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Frequency Response



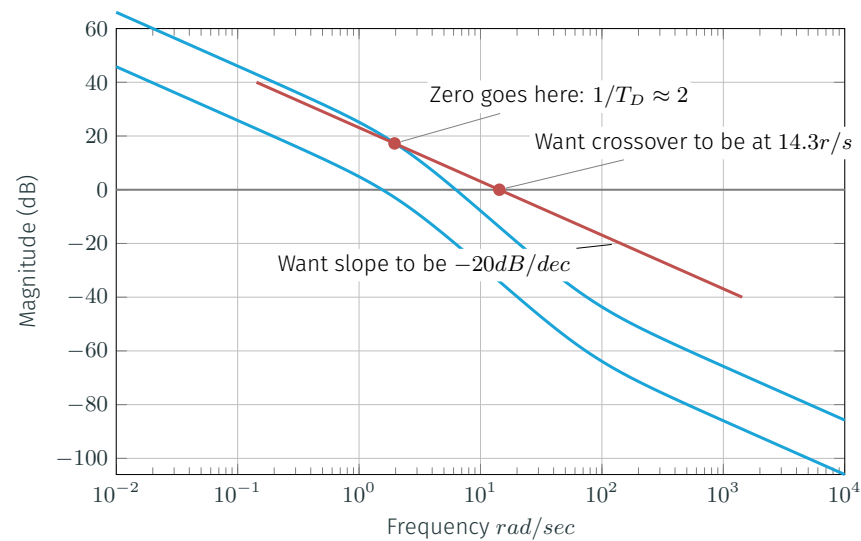
39

Frequency Response



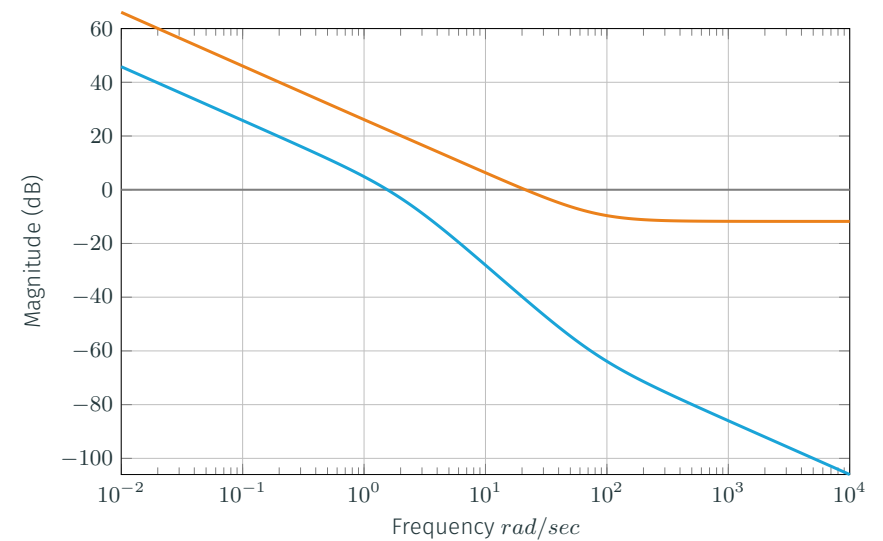
39

Frequency Response



39

Frequency Response



39

Example - PD Controller

$$G(s) = 0.05 \frac{80 - s}{s(s + 2.05)}$$

Design a PD controller such that:

- Steady-state error to step input is zero
- Track ramp with steady-state error less than 0.05
- Closed-loop step response with time-constant less than 0.07s
- Phase margin greater than 60°

Our final controller is:

$$K(s) = 10.5 \cdot (1 + s/2)$$

Example 9.10

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Lead Compensator

Benefit

- Increase the phase near the crossover frequency

Downside

- Increases the gain at high-frequencies
→ Increases sensitivity to noise and unmodeled dynamics

Lag Compensator

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Phase Lag Compensator

Lead Compensator

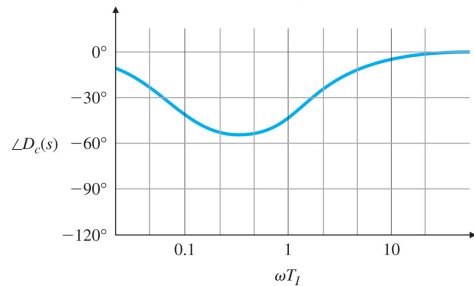
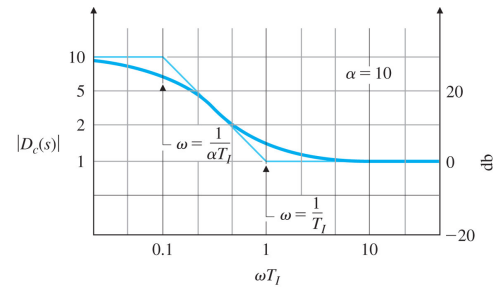
$$D_c(s) := \alpha \frac{T_I s + 1}{\alpha T_I s + 1} \quad \alpha > 1$$

Within interval of interest

- Phase decreased by up to 90°
- Gain increased below frequency $1/T_I$

Utility:

- Increase gain at low frequencies to reduce steady-state errors



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Phase Lag Compensator

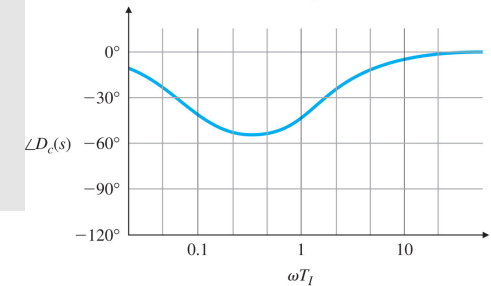
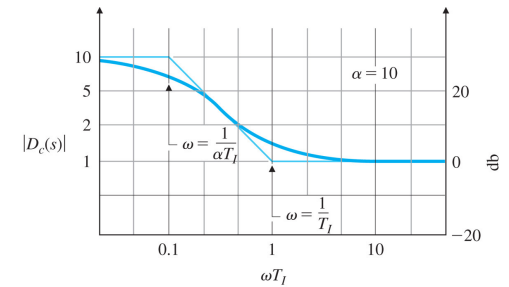
Goal

Increase low-frequency gain, without impacting transient behaviour

Idea

- Set break frequency $\frac{1}{T_I}$ below the crossover frequency, to not impact transient behaviours
- Choose α to give desired steady-state behaviour

$$\lim_{s \rightarrow 0} \alpha \frac{T_I s + 1}{\alpha T_I s + 1} = \alpha$$



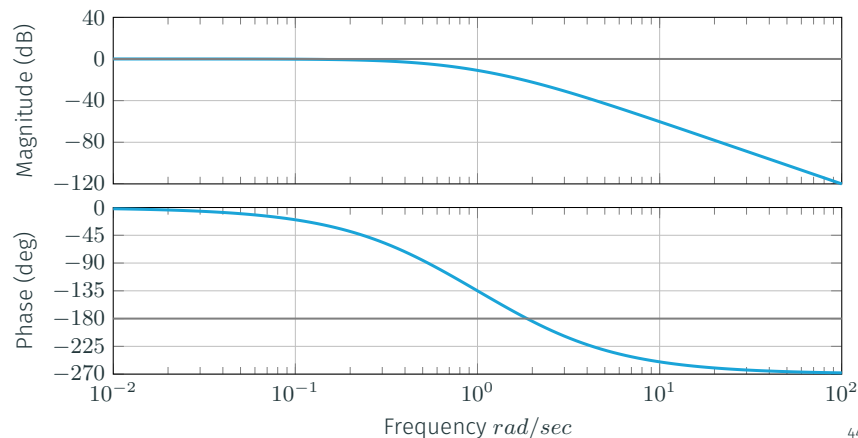
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43

Example

$$G(s) = \frac{1}{(\frac{1}{0.5}s + 1)(s + 1)(\frac{1}{2}s + 1)}$$

Design a lag compensator to achieve a phase margin of at least 40° and a steady-state error with respect to a step input better than 10%.

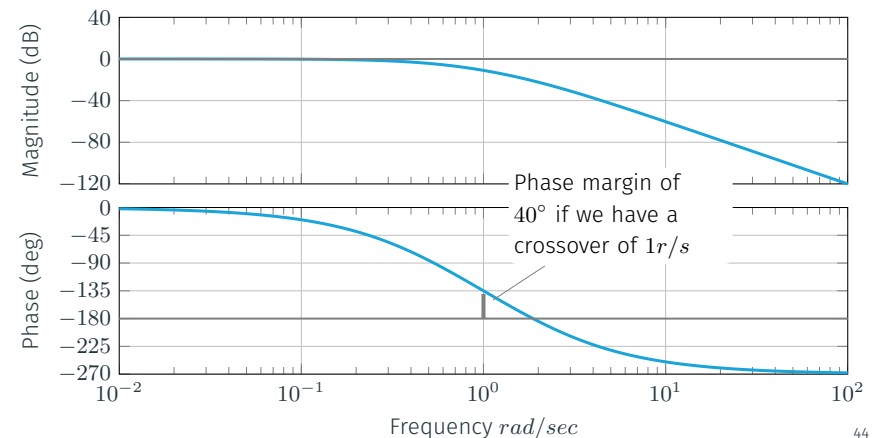


44

Example

$$G(s) = \frac{1}{(\frac{1}{0.5}s + 1)(s + 1)(\frac{1}{2}s + 1)}$$

Design a lag compensator to achieve a phase margin of at least 40° and a steady-state error with respect to a step input better than 10%.

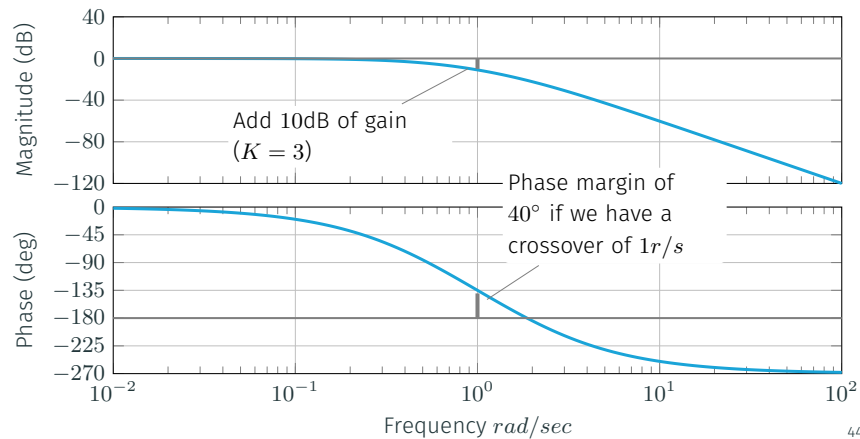


44

Example

$$G(s) = \frac{1}{(\frac{1}{0.5}s + 1)(s + 1)(\frac{1}{2}s + 1)}$$

Design a lag compensator to achieve a phase margin of at least 40° and a steady-state error with respect to a step input better than 10%.

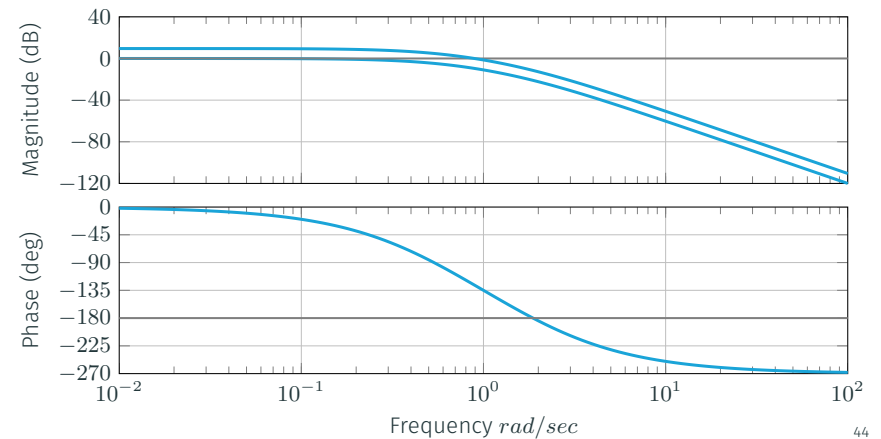


44

Example

$$G(s) = \frac{1}{(\frac{1}{0.5}s + 1)(s + 1)(\frac{1}{2}s + 1)}$$

Design a lag compensator to achieve a phase margin of at least 40° and a steady-state error with respect to a step input better than 10%.

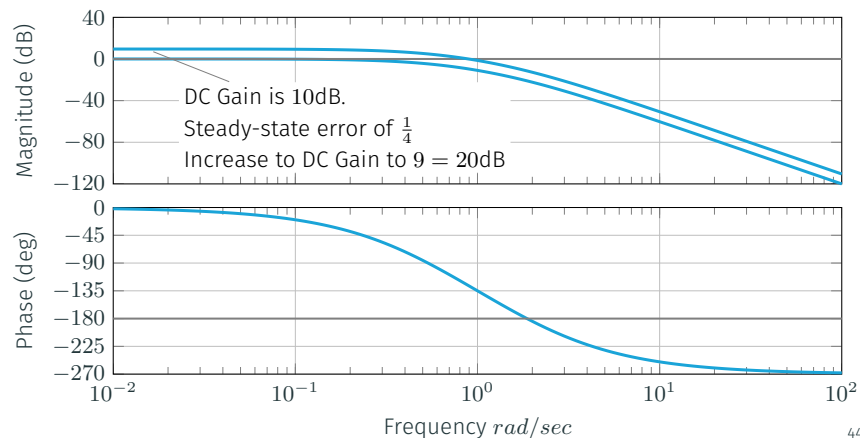


44

Example

$$G(s) = \frac{1}{(\frac{1}{0.5}s + 1)(s + 1)(\frac{1}{2}s + 1)}$$

Design a lag compensator to achieve a phase margin of at least 40° and a steady-state error with respect to a step input better than 10%.

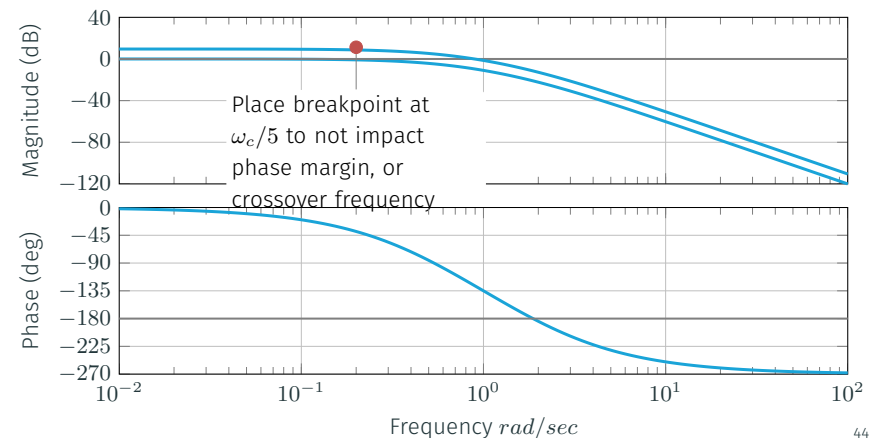


44

Example

$$G(s) = \frac{1}{(\frac{1}{0.5}s + 1)(s + 1)(\frac{1}{2}s + 1)}$$

Design a lag compensator to achieve a phase margin of at least 40° and a steady-state error with respect to a step input better than 10%.

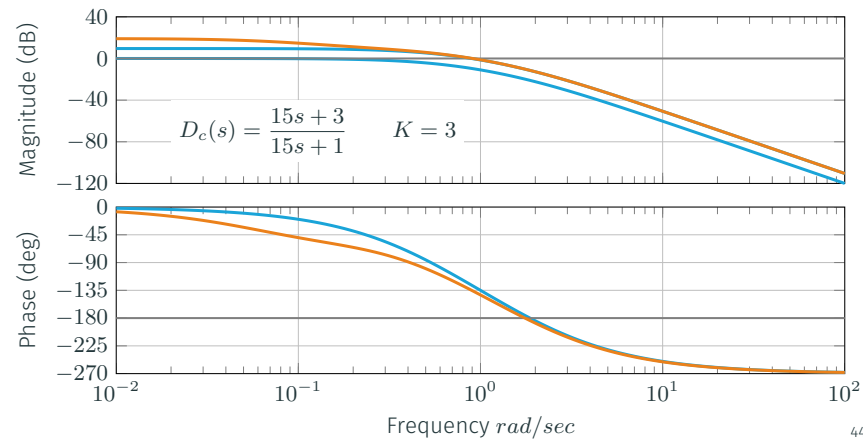


44

Example

$$G(s) = \frac{1}{(\frac{1}{0.5}s + 1)(s + 1)(\frac{1}{2}s + 1)}$$

Design a lag compensator to achieve a phase margin of at least 40° and a steady-state error with respect to a step input better than 10%.

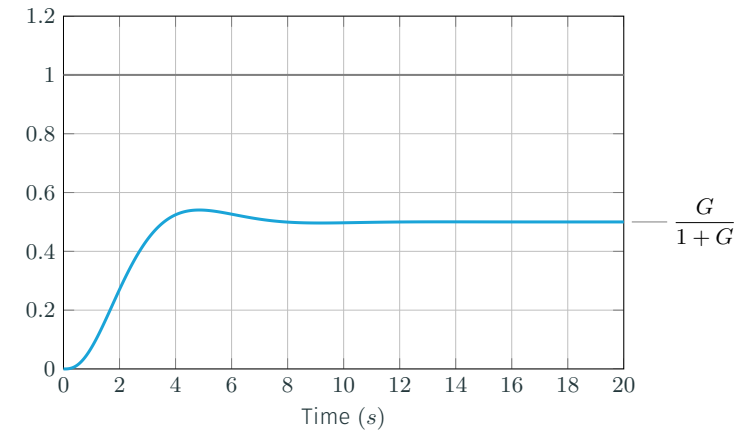


44

Time Response

$$G(s) = \frac{1}{(\frac{1}{0.5}s + 1)(s + 1)(\frac{1}{2}s + 1)}$$

Design a lag compensator to achieve a phase margin of at least 40° and a steady-state error with respect to a step input better than 10%.

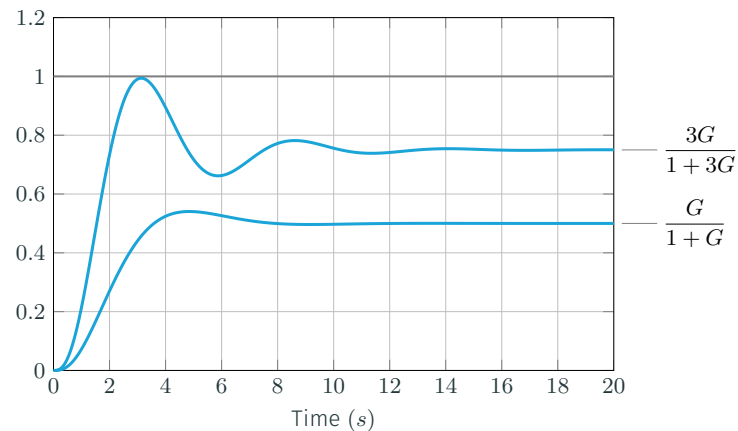


45

Time Response

$$G(s) = \frac{1}{(\frac{1}{0.5}s + 1)(s + 1)(\frac{1}{2}s + 1)}$$

Design a lag compensator to achieve a phase margin of at least 40° and a steady-state error with respect to a step input better than 10%.

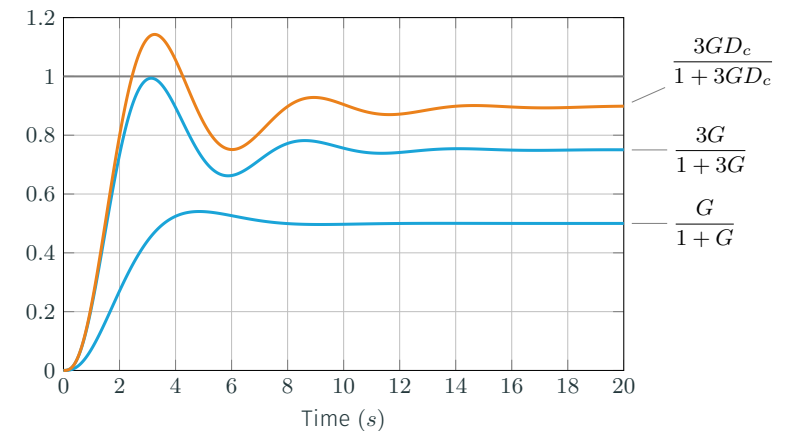


45

Time Response

$$G(s) = \frac{1}{(\frac{1}{0.5}s + 1)(s + 1)(\frac{1}{2}s + 1)}$$

Design a lag compensator to achieve a phase margin of at least 40° and a steady-state error with respect to a step input better than 10%.



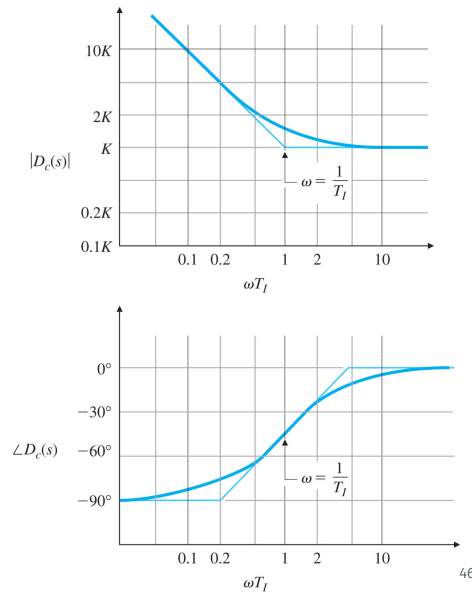
45

Relation to PI Controller

What happens when $\alpha \rightarrow \infty$?

$$\begin{aligned} K D_c(s) &= K \alpha \frac{T_I s + 1}{\alpha T_I s + 1} \\ &= K \frac{T_I s + 1}{T_I s + \frac{1}{\alpha}} \\ &= K \left(1 + \frac{1}{T_I s} \right) \end{aligned}$$

We see that a lag compensator is a PI controller with $\alpha = \infty$



Example

$$G(s) = 0.05 \frac{80 - s}{s + 2.05}$$

Specifications:

- Zero steady-state error to step command
- Phase margin greater than 60°
- Closed-loop time constant of $1/\omega_c = 0.07s$ ($\omega_c = 14.3\text{rad/s}$)

Example 9.11

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Lead/Lag Compensators

- | | |
|------------------|--|
| Lead compensator | Adds phase at crossover frequency to improve margins
Impacts frequencies above the breakpoint |
| Lag compensator | Adds gain at low frequency to improve steady-state response
Impacts frequencies below the breakpoint |

We are free to use lead and lag filters in combination, without them impacting each other, often called lead-lag filters.

Lead $\rightarrow$ PD controller

Lag $\rightarrow$ PID controller

Lead/Lag Compensator

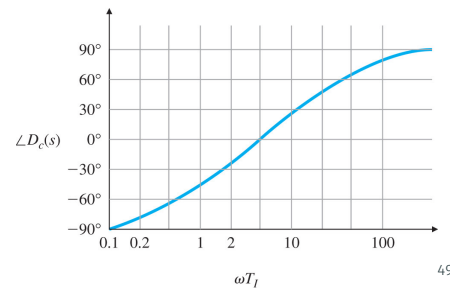
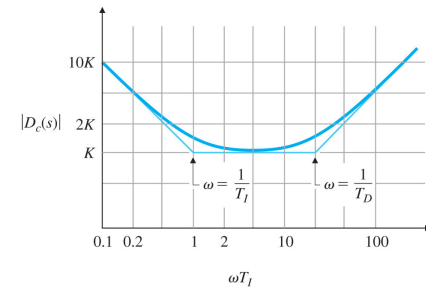
A PID controller is a lead/lag filter

$$D_c(s) = K (T_D s + 1) \left(1 + \frac{1}{T_I s} \right)$$

48

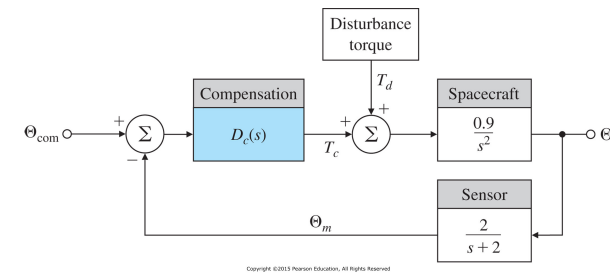
PID Lead/Lag Filter

Frequency response of PID compensator for $\frac{T_I}{T_D} = 20$



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Satellite Stabilization Problem



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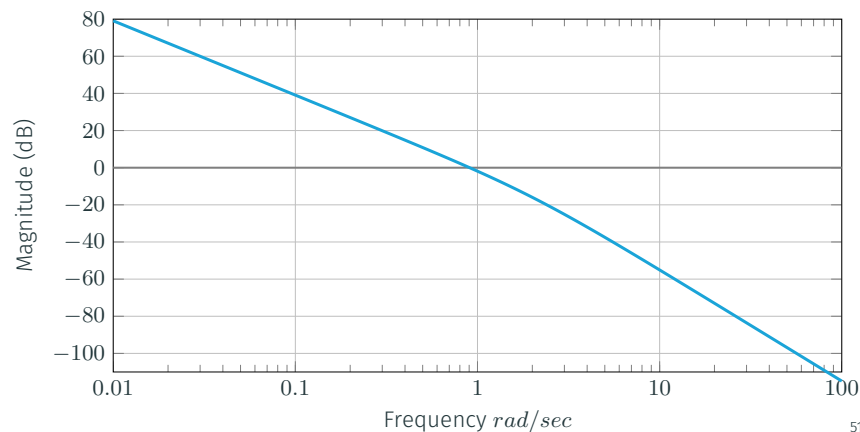
Design a PID controller for

- Zero steady-state error in response to a step disturbance torque
- A phase margin of approximately 60°
- As high a bandwidth as possible

50

Example

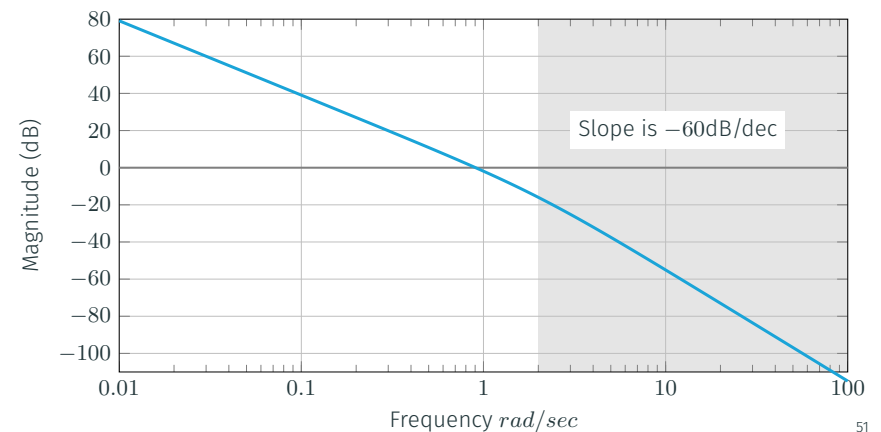
- Zero steady-state error in response to a step disturbance torque
- A phase margin of approximately 60°
- As high a bandwidth as possible



51

Example

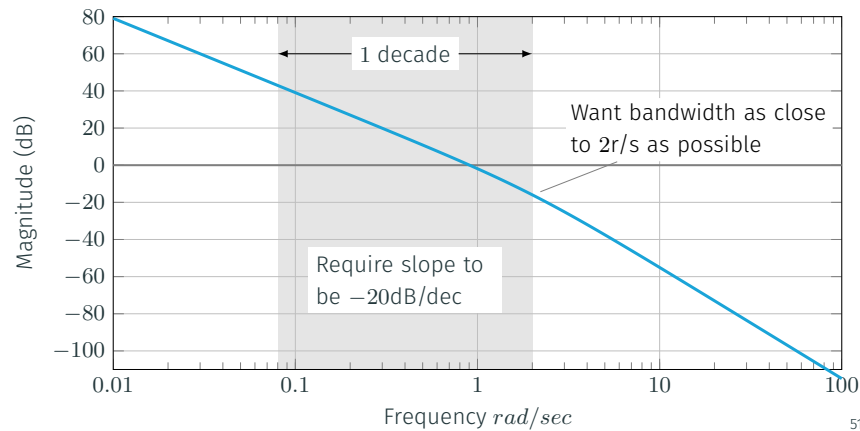
- Zero steady-state error in response to a step disturbance torque
- A phase margin of approximately 60°
- As high a bandwidth as possible



51

Example

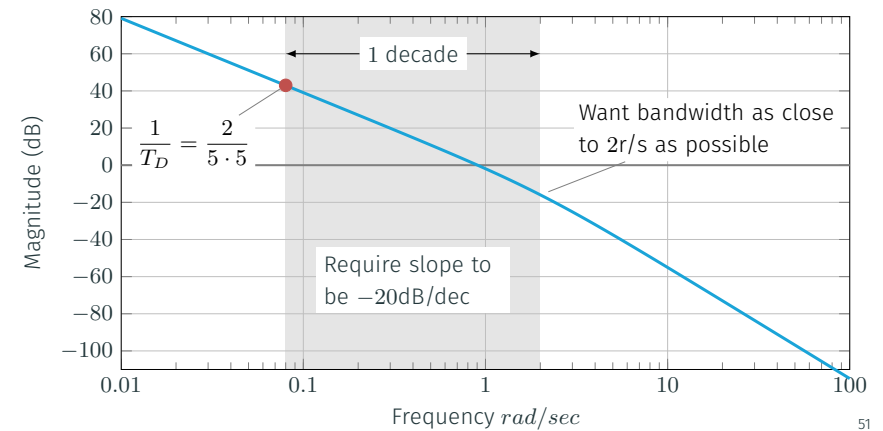
- Zero steady-state error in response to a step disturbance torque
- A phase margin of approximately 60°
- As high a bandwidth as possible



51

Example

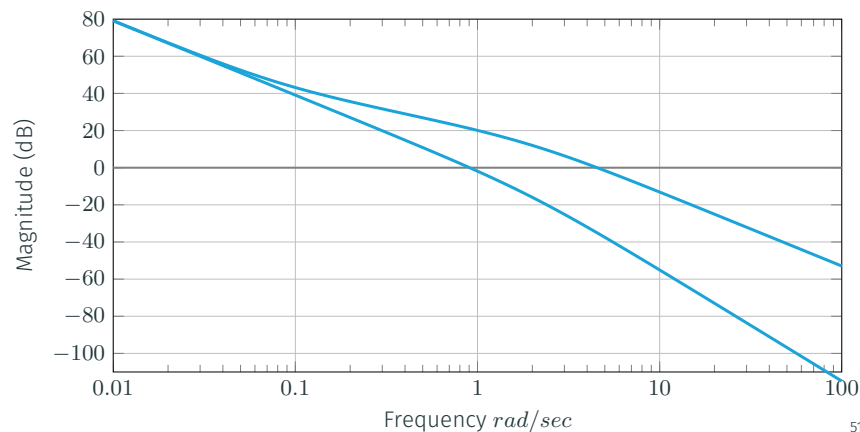
- Zero steady-state error in response to a step disturbance torque
- A phase margin of approximately 60°
- As high a bandwidth as possible



51

Example

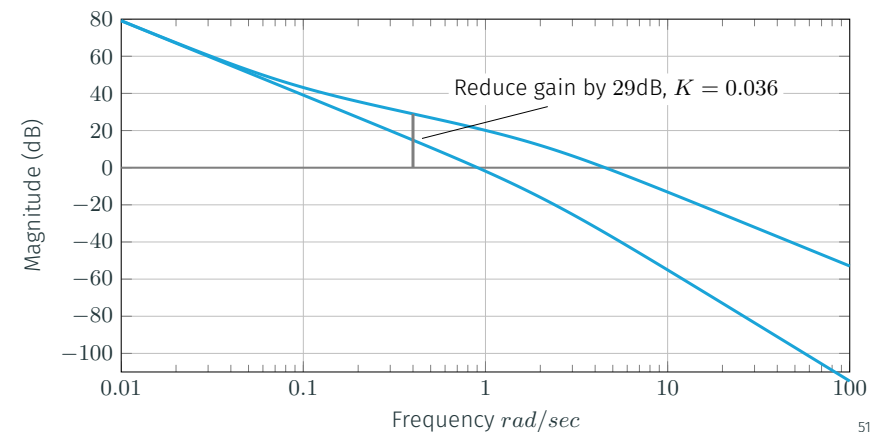
- Zero steady-state error in response to a step disturbance torque
- A phase margin of approximately 60°
- As high a bandwidth as possible



51

Example

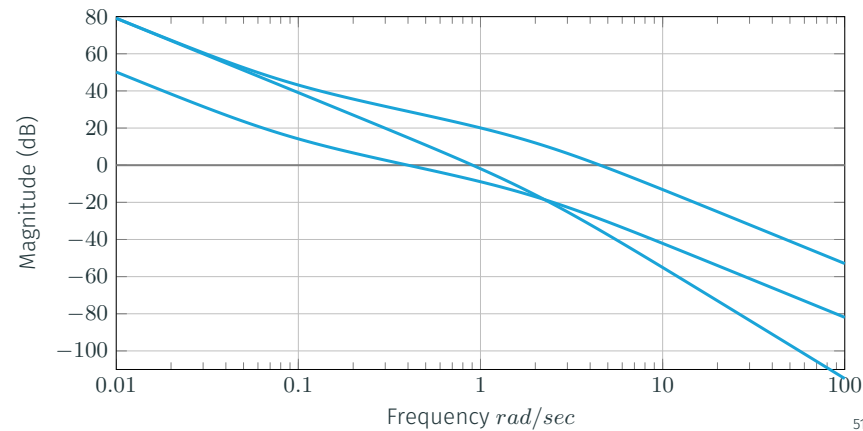
- Zero steady-state error in response to a step disturbance torque
- A phase margin of approximately 60°
- As high a bandwidth as possible



51

Example

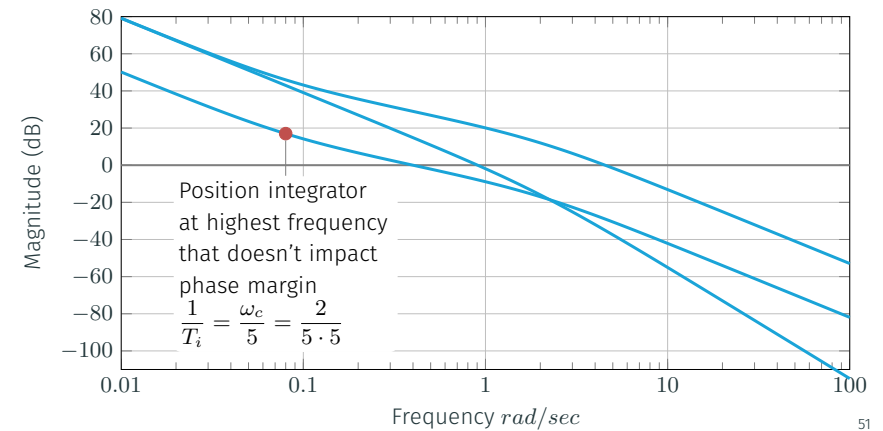
- Zero steady-state error in response to a step disturbance torque
- A phase margin of approximately 60°
- As high a bandwidth as possible



51

Example

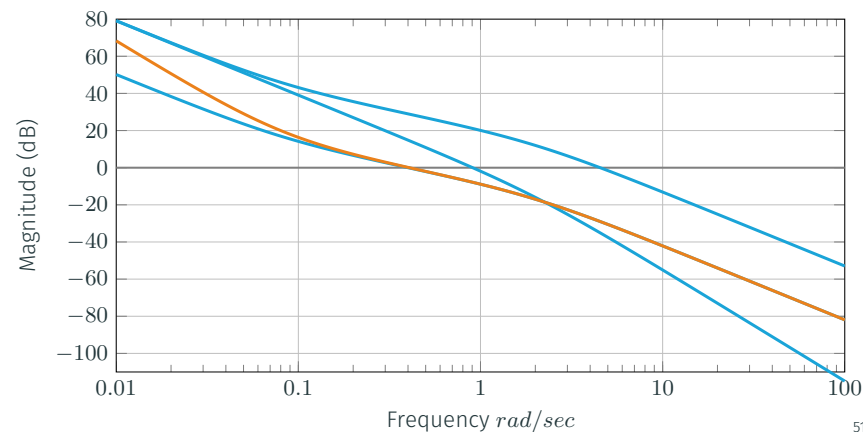
- Zero steady-state error in response to a step disturbance torque
- A phase margin of approximately 60°
- As high a bandwidth as possible



51

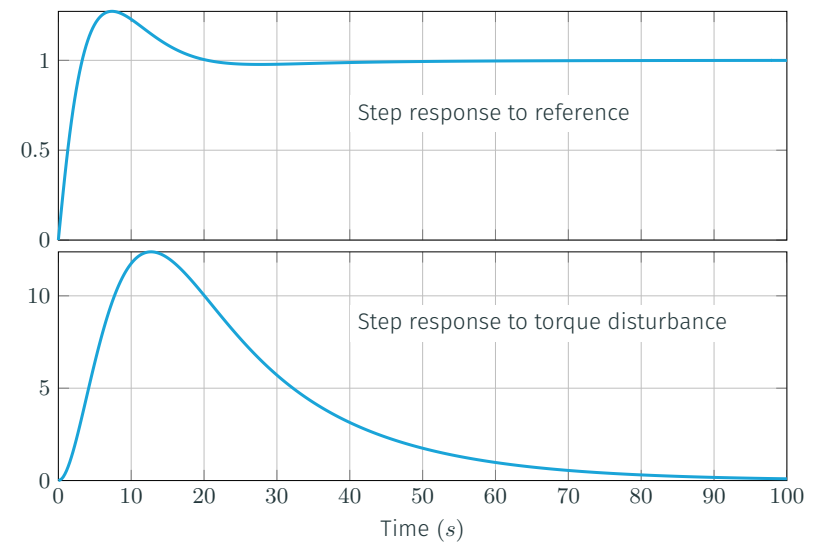
Example

- Zero steady-state error in response to a step disturbance torque
- A phase margin of approximately 60°
- As high a bandwidth as possible



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Time Response



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Summary - Loop Shaping

Idea Can relate the shape of the frequency response of the open-loop system to the closed-loop sensitivity and complementary sensitivity functions

Goal

- KG large for small ω (Steady-state error)
- KG small for large ω (Modeling errors, etc)
- Crossover frequency chosen according to desired closed-loop bandwidth
- Good stability margins / slope of KG equal to -20dB/dec at crossover

Lead compensator

- Increase slope by 20dB/dec in frequency range
- Use to increase slope / phase near crossover frequency
- PD controller is a lead compensator

Lag compensator

- Use to increase gain at low frequencies
- Decreases slope by 20dB/dec / decreases phase in frequency range
- PI controller is a lag compensator

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A Real System Design

Atomic Force Microscope

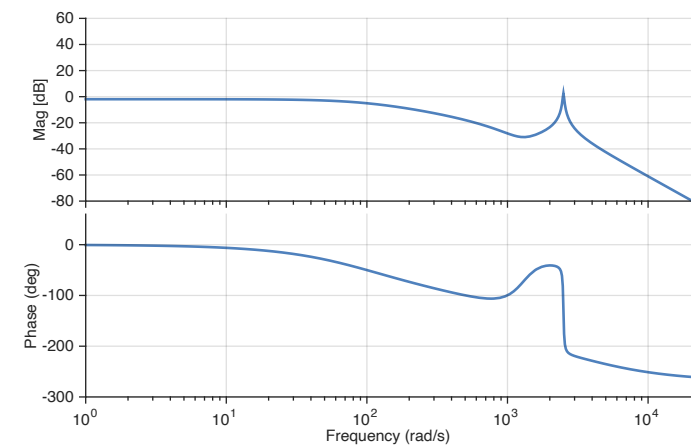
$$G(s) = 8.88 \cdot 10^8 \frac{s^2 + 780s + 1.69 \cdot 10^6}{(s + 3000)(s + 1000)(s + 100)(s^2 + 50s + 6.25 \cdot 10^6)}$$

Goals

- Track ramp inputs
- Reduce sensitivity to noise at resonant frequency
- Minimize response time

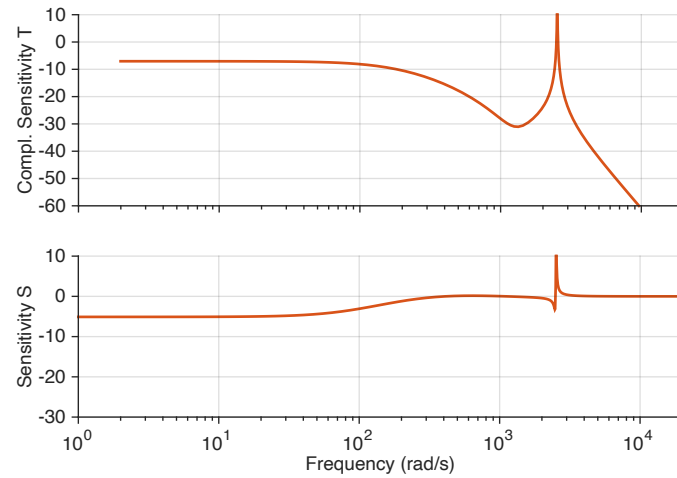
54

Frequency Response



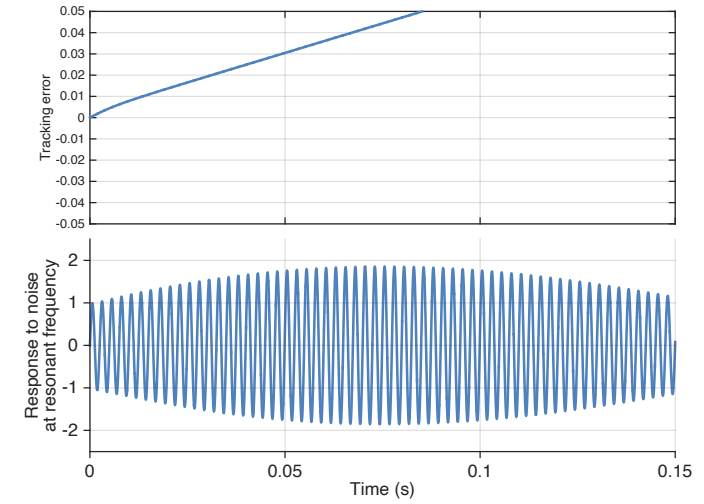
55

Closed-Loop Sensitivity



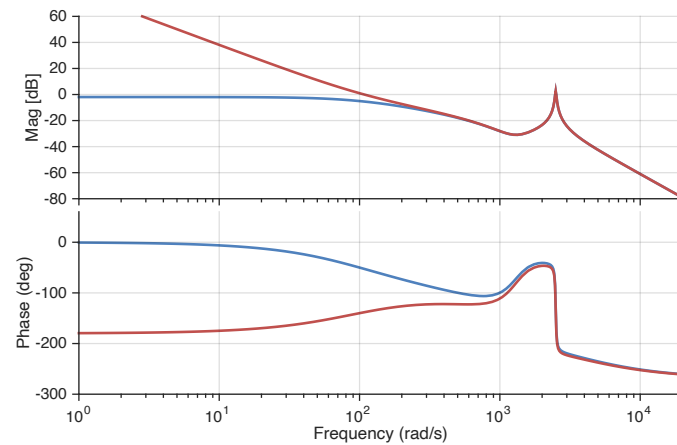
56

Step Response & Resonance Disturbance Rejection



57

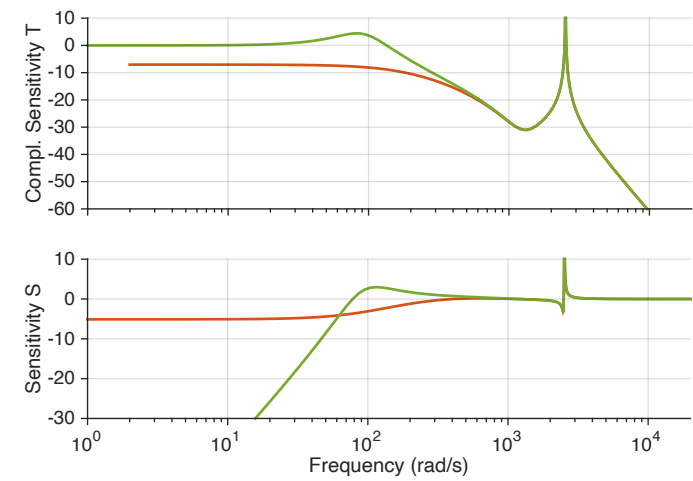
Frequency Response



Track ramps $\rightarrow$ Add two integrators $\left(1 + \frac{1}{T_i s}\right)^2$

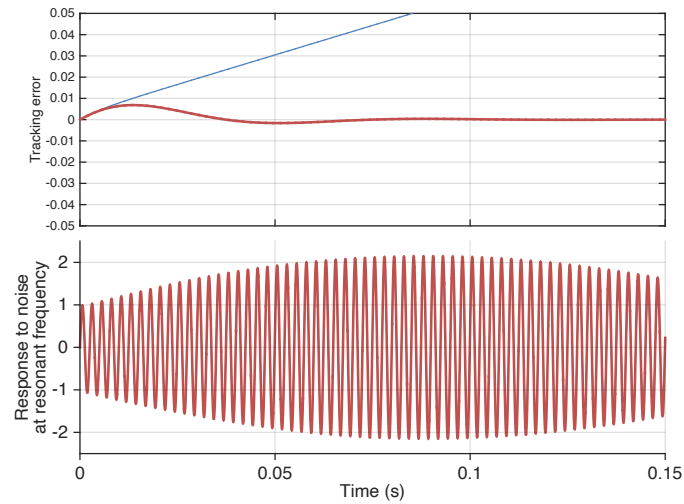
58

Closed-Loop Sensitivity



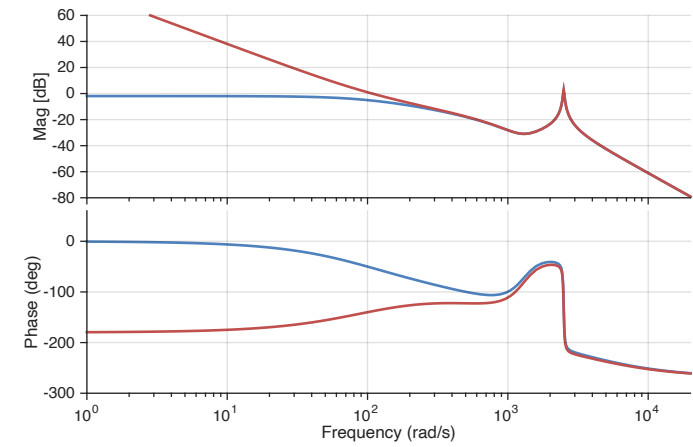
59

Step Response & Resonance Disturbance Rejection



60

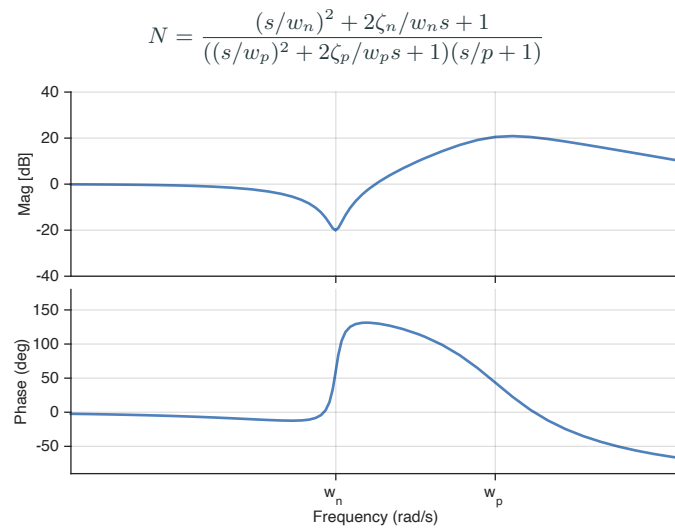
Frequency Response



Reducing sensitivity at resonance, requires a *high* gain.
Problem: drop of 180 degrees of phase.

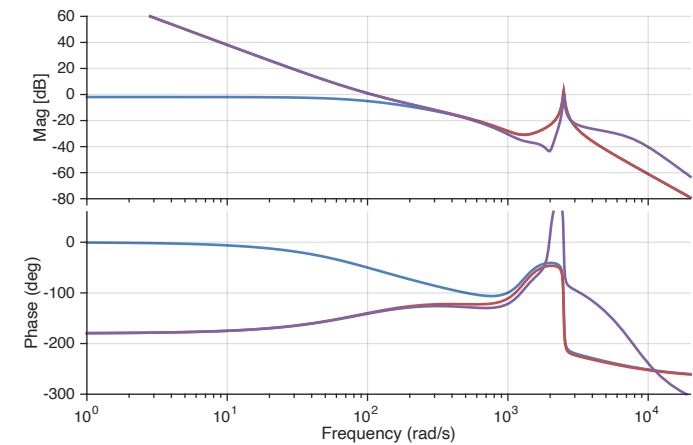
61

Notch Filter



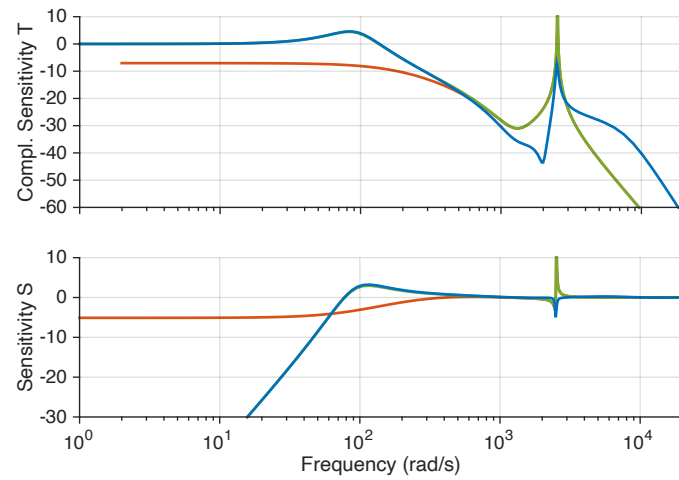
62

Frequency Response



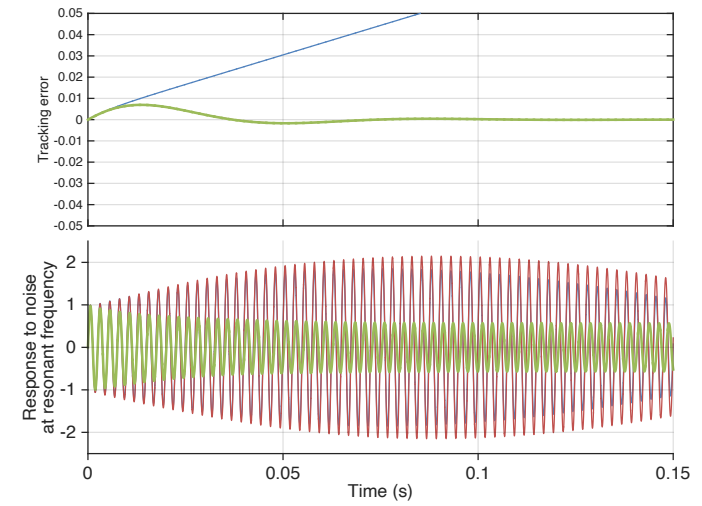
63

Closed-Loop Sensitivity



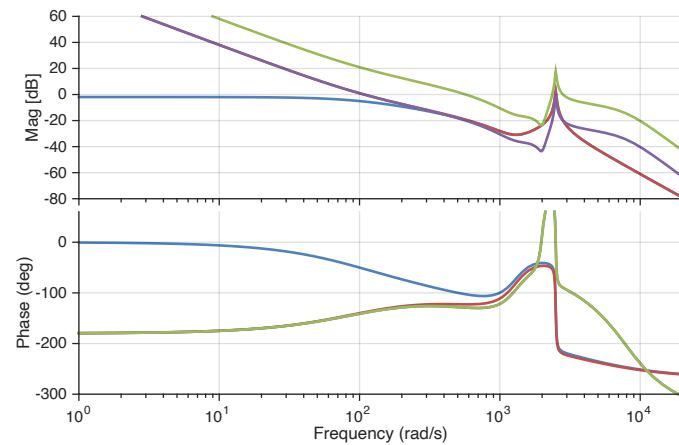
64

Step Response & Resonance Disturbance Rejection



65

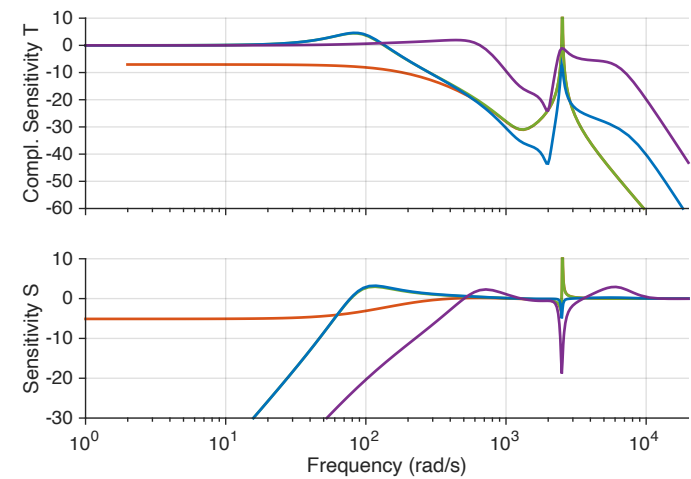
Frequency Response



Increase gain to get best tracking performance.

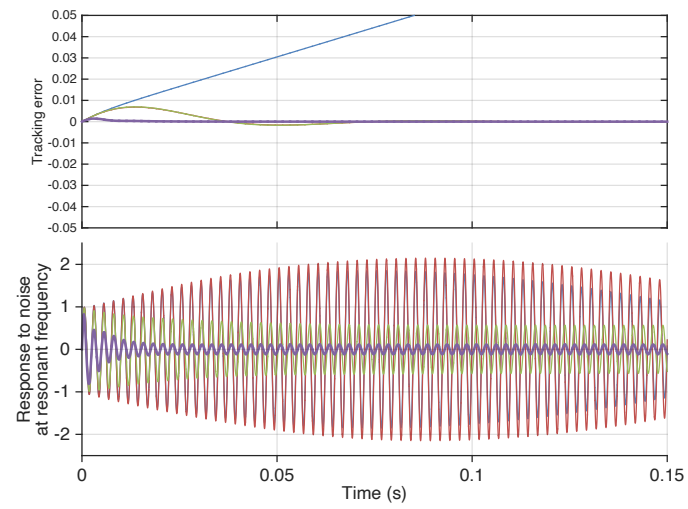
66

Closed-Loop Sensitivity



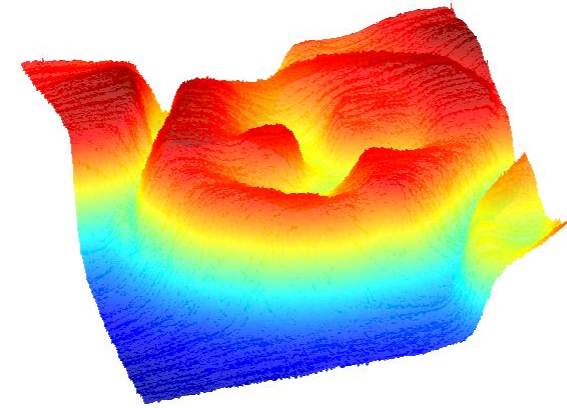
67

Step Response & Resonance Disturbance Rejection



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Result of a Scan



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